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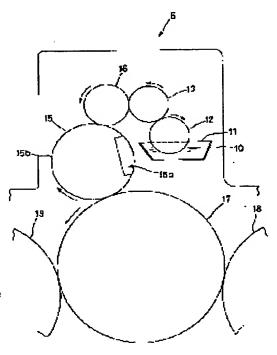
#### (54) COATING DEVICE FOR PRINTER

(57)Abstract:

PROBLEM TO BE SOLVED: To reduce a varnish

unevenness.

SOLUTION: A varnish 11 is lifted by a ductor roller 12 dipped in the varnish 11 of a varnish pan 10, and a uniform varnish film is formed on a metering roller 13 opposite contacted with the varnish 11. The varnish is transferred to a rubber cylinder 15 oppositely contacted with a distributing roller 16 via the roller 16 oppositely contacted with the roller 13. A sheet is gripped by a gripper of an impression cylinder 17 and conveyed, and its printing surface is coated with the varnish when passing an opposite contact point with the cylinder 15. The cylinder 15 is rotatably driven via a rubber cylinder gear drivably coupled to a prime mover side, and the roller 16 is rotatably driven via a clutch gear engaged with the cylinder gear. A diameter of a pitch circle of the cylinder gear is set to the same as that of the rubber cylinder, and a diameter of a pitch circle of the clutch gear is set to smaller diameter than the diameter of the dipping roller.



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#### CLAIMS

[Claim(s)]

[Claim 1] Coating equipment of the printing machine characterized by said thing [ having stuck and having made peripheral speed of a roller quicker than the peripheral speed of said drum ] in the coating equipment of the printing machine which was supported free [ the rotation which opposite-\*\* on the drum on which it was prepared in coating equipment, and it was supported free / rotation / and the notch was prepared in the peripheral surface, and this drum, and supplies varnish to this drum ], and which stuck and was equipped with the roller.

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#### DETAILED DESCRIPTION

## [Detailed Description of the Invention] [0001]

[Field of the Invention] This invention is prepared between the printing unit of a printing machine, and delivery equipment, and relates to the coating equipment of the printing machine which applies varnish to the printing side of the paper after printing, and is discharged to delivery equipment.

#### [0002]

[Description of the Prior Art] In this kind of printing machine, coating equipment is formed between a printing unit and delivery equipment, and there are some which incorporated coating equipment in the printing machine in order to reduce presswork. It opposite—\*\* with the blanket cylinder with which the blanket was twisted, and this blanket cylinder, and rotates and sticks in the same direction as a blanket cylinder with the same peripheral speed in this opposite \*\*\*\*, and this coating equipment is equipped with the roller and this metering roller that rotates to a roller and an opposite direction by sticking, opposite—\*\*(ing) on a roller and sticking to it in this opposite \*\*\*\*. The rotation drive of the blanket cylinder is carried out through the blanket cylinder gear by which drive connection was carried out with the original working side, it sticks, and with a blanket cylinder gear, the roller got into gear, and is stuck, and the rotation drive is carried out through the roller gear. Moreover, the rotation drive of the metering roller is carried out with the former roller which pulls up the varnish of a varnish brake shoe by making a motor different from a original working side into a driving source.

[0003]

[Problem(s) to be Solved by the Invention] the vice with which the peripheral surface of a blanket cylinder adds the both ends of a blanket — it is divided into the notch by which equipment was contained, and the so-called measuring area where the blanket was rolled, and it sticks, and a roller is with the time of opposite-\*\*(ing) to the time of opposite-\*\*(ing) to measuring area, and the notch, and the condition of having escaped with the condition that nip pressure was added is repeated. That is, braking is applied by frictional force with the metering roller which is rotating to hard flow, while a rotation drive is carried out by frictional force with the peripheral surface of the blanket cylinder which rotates in the same direction in opposite \*\*\*\* while sticking and opposite-\*\*(ing) the roller with the measuring area of a blanket cylinder and opposite-\*\*(ing) with the notch according to it. therefore, a blanket cylinder — sticking — management of the nip pressure between rollers — or if managing [ of the diameter of a cut of a blanket cylinder ] the path of a blanket cylinder worsens greatly, since it will stick with the peripheral surface of a blanket cylinder in measuring area and frictional force with the peripheral surface of a roller will become large, it sticks and the rotation drive of the roller may be carried out by only this frictional force instead of a rotation drive from a gear

[0004] In this condition, since it will be in the condition that stuck to the blanket cylinder gear and the roller gear preceded, it will be in the condition that stuck from the blanket cylinder gear and the rotation drive to a roller gear was intercepted. Since stick with the peripheral surface of a blanket cylinder in a notch, and frictional force with the peripheral surface of a roller is almost lost, it sticks according to frictional force on the other hand, and the rotation drive of the roller

is not carried out and it slows down according to the frictional force between metering rollers, it sticks, and the rotation drive of a roller sticks with a blanket cylinder gear, and is performed by the rotation transfer with a roller gear. Thus, on the boundary of the measuring area of a drum, and a notch, since it sticks from a blanket cylinder gear, and the rotation drive to a roller gear is intercepted or it is connected, it sticks by the backlash between mutual gears, and big rotation fluctuation occurs on a roller. For this reason, the shock eye of a band-like striped pattern was formed in the direction which intersects perpendicularly with paper with the printing direction, and this caused varnish nonuniformity.

[0005] The place which this invention is made in view of the above-mentioned conventional problem, and is made into the purpose is to offer the coating equipment of the printing machine which reduced varnish nonuniformity.

[0006]

[Means for Solving the Problem] the coating equipment of the printing machine which this invention was prepared in coating equipment in order to attain this purpose, and was supported free [ rotation ] and which stuck and was equipped with a roller and this drum on which it stuck, and opposite—\*\*(ed) with the roller, varnish was supplied, it was supported free [ rotation ], and the notch was prepared in the peripheral surface — setting — said — it sticks and peripheral speed of a roller is made quicker than the peripheral speed of said drum. Therefore, since it sticks and a skid occurs between the peripheral surface of a roller, and the peripheral surface of a drum when it sticks and the peripheral surface of a roller always precedes rather than the peripheral surface of a drum Even if it sticks with the peripheral surface of a drum in the measuring area of a drum and frictional force with the peripheral surface of a roller becomes large, it will stick so that it may follow and stick to rotation of a drum and may not accelerate the peripheral speed of a roller, and the peripheral speed of a roller will not change on the measuring area of a drum, and the boundary of a notch.

[0007]

[Embodiment of the Invention] Hereafter, the gestalt of operation of this invention is explained based on drawing. They are the external view of the whole printing machine which drawing 1 requires for this invention, and the sectional view in which drawing 2 is the same, and the side elevation of coating equipment and drawing 3 are the same, and having shown the supporting structure of a former roller and a metering roller. It is a model Fig. for the sectional view and drawing 5 which drawing 4 similarly stuck with the blanket cylinder, and showed the supporting structure of a roller being the same, and the side elevation of an important section and drawing 6 being the same, and explaining a rotation drive system. Drawing and this drawing (c) showing a rotation transfer path [ in / in drawing for the side elevation of an important section for drawing 7 to explain a rotation drive similarly and this drawing (a) to explain attachment and detachment of a blanket cylinder and this drawing (b) / a blanket cylinder arrival condition ] are drawing showing the rotation transfer path in a blanket cylinder decondition.

[0008] In drawing 1, the sheet rotary press in which the whole is shown with a sign 1 is equipped with the varnish coating unit 5 as a coater unit prepared between feed equipment 2, the printing unit 3 of four colors, delivery equipment 4, and these printing units 3 and delivery equipment 4. In drawing 2, the varnish brake shoe 10 by which varnish 11 was stored is formed in the varnish coating unit 5, it is immersed in the varnish 11 of this varnish brake shoe 10 in the former roller 12, and the metering roller 13 is opposite—\*\*(ed) by this former roller 12. The varnish which is the same direction, and the rotation drive was carried out with the same peripheral speed in opposite \*\*\*\* mutual as a driving source, and was able to pull up the motor 20 which mentions these former roller 12 and the metering roller 13 later from the varnish brake shoe 10 with the former roller 12 is formed in the front face of the metering roller 13 as uniform varnish film by passing nip with the metering roller 13.

[0009] the vice which 15 is a blanket cylinder, and adds the both ends of a blanket while the blanket (not shown) is twisted around the peripheral surface — notch 15a for containing equipment (not shown) is prepared, and the peripheral surface of this blanket cylinder 15 is divided into notch 15a and measuring—area 15b around which the blanket was twisted. A rotation drive is carried out through the blanket cylinder gear 60 by which drive connection was carried

out with the original working side and which is mentioned later, the clutch gear 59 which sticks to this blanket cylinder gear 60, and a roller 16 mentions later meshes through the middle gears 59A and 59B, a blanket cylinder 15 is stuck and the roller 16 is rotating it in the same direction in a blanket cylinder 15 and mutual opposite \*\*\*\*.

[0010] The paper added to the pawl of the delivery drum 18 of a printing unit 3 is changed and conveyed in addition to the pawl of this impression cylinder 17, a blanket cylinder 15 is an impression cylinder which opposite—\*\*, when passing an opposite contact with a blanket cylinder 15, varnish is applied to a printing side, and 17 is again added to the pawl of the delivery drum 19 of delivery equipment 4. The paper which is added to the pawl of the delivery drum 19 and is conveyed is changed in addition to the pawl (illustration [ which ] is omitted) of \*\*\*\* of a delivery chain, it is conveyed with the delivery chain it runs, and fall loading is carried out on \*\*\*\*\*\* of delivery equipment 4.

[0011] In drawing 3, 20 is the motor fixed to the frame 21 through the bracket 22, the gearbox 24 directly linked through coupling 23 is directly linked with a motor shaft, and this gearbox 24 is being fixed to the frame 21 through the bracket 25. The drive gear shaft 26 by which intersected perpendicularly with the motor direct connection shaft in this gearbox 24, and drive connection was carried out with this by the internal bevel gear is supported to revolve, and the drive gear 28 supported free [ rotation ] is fixed to revolve with the stud 27 which projects from a frame 21 by this drive gear shaft 26.

[0012] The gear shaft 30 supported for the middle gear 29 which meshes with the drive gear 28, enabling free rotation is supported to revolve through bearing 31, and the end of the former roller 12 is supported by the frame 21 free [ rotation ] at the inside protrusion bearing 32 of the frame 21 of this gear shaft 30. The former roller gear 33 is fixed to the flange of the axis end of the former roller 12, and it is supported by this former roller gear 33 free [ the rotation to the bearing 36 by which these gears 34 and 35 were fixed to revolve by the both ends of the gear shaft 37 by transmitting rotation of the middle gear 29 through gears 34 and 35, and the gear shaft 37 was attached in the frame 21 ].

[0013] Between the bearing 32 of said gear shaft 30, and bearing 31, the roller arm 39 is \*\*\*\* (ed) through thrust bearing, and the T character arm 40 is pivoted in one free end section of this roller arm 39 free [ rocking ] through the pin 41. As the sign t1 in drawing shows, the bearing 42 to which eccentricity of the bearing was carried out is being fixed to one free end section of this T character arm 40 free [ rotation accommodation ], this bearing 42 is made to opposite-\*\* a peripheral surface on the former roller 12, and the metering roller 13 which has an elastic front face is supported to revolve.

[0014] By meshing with the former roller gear 33 the metering roller gear 43 fixed to that axis end section, a rotation drive is carried out, and by loosening a bolt and rotating bearing 42, this metering roller 13 is constituted so that the nip pressure to the former roller 12 may be adjusted. 44 is the cam equipped with the major diameter and the narrow diameter portion, it is fixed to the both frame 21 contiguity section of the cam shaft 45 passed over between the frames 21 (not shown [ one frame 21 ]) on either side, and as the sign t2 in drawing showed to the free end section of the T character arm 40, when it was pivoted in this cam side free [ eccentric accommodation ], 46 is opposite—\*\*(ing).

[0015] The spring shaft 48 in which rotation accommodation is free is pivoted and supported to revolve by the T character arm 40 in the end at the stud 47 implanted in the frame 21, and the T character arm 40 is having the rotation force energized in the direction which carries out the pressure welding of the roller 46 to a cam 44 by the resiliency of the compression coil spring 49 on the spring shaft 48. It projects and the rod (all omit illustration) of an air cylinder is pivoted in an edge through a lever, and a cam shaft 45 rotates by [ which projected from the frame 21 of another side which omitted illustration of a cam shaft 45 ] making a rod move, and it is constituted so that the metering roller 13 may be detached and attached to the former roller 12 through a roller 46 and the T character arm 40.

[0016] 50 is a clutch shaft, it is supported to revolve with the bearing 51 fixed to the frame 21, and the bracket 52 which projects from a frame 21, and the gear 53 which meshes with said drive gear 28 on this clutch shaft 50, and transmits the rotation by the side of a motor 20 to a

clutch shaft 50 is fixed to revolve. The clutch gear 55 fixed to the one way clutch 54 on a clutch shaft 50 was stuck, it was fixed to the axis end section of a roller 16, and it was stuck, and meshes with the roller gear 56 (refer to drawing 4). It is constituted so that it sticks, and a roller 16 may serve as a driving side—ed, only the rotation by the side of a motor 20 may stick and it may be transmitted to a roller 16 with this one way clutch 54.

[0017] As shown in drawing 4, it stuck, the axis end section of a roller 16 was also equipped with the one way clutch 58, and it has geared through the drum [ which was fixed to this ] gear 60 by which the clutch gear 59 as a gear by the side of a roller was fixed to the axis end section of said blanket cylinder 15 by sticking, and the middle gears 59A and 59B. It is constituted so that it sticks, and a roller 16 side may turn into a driving side—ed, only the rotation by the side of a blanket cylinder 15 may stick and it may be transmitted to a roller 16 with this one way clutch 58. Thus, although it sticks and the rotation drive of the roller 16 is carried out through one way clutches 54 and 58 from both by the side of a motor 20 and a blanket cylinder 15, it is constituted so that both one way clutches 54 and 58 may not transmit rotation to coincidence, the one way clutch of quick either of the driving—side rotations may transmit rotation and the later one may race.

[0018] That is, in drawing 7 (a), a continuous-line location sticks a blanket cylinder 15, and the rotation [ in / it wears is a location and / in this drawing (b) / this blanket cylinder arrival condition ] transfer path to a roller 16 and an impression cylinder 17 is shown. At this time, the former roller 12 and the metering roller 13 were driven by the motor 20, were stuck, and on the other hand, the roller 16 was driven through the clutch 58 from the impression cylinder 17 and blanket cylinder 15 side, and they have raced the one way clutch 54 of another side in this condition. Moreover, in this drawing (a), the location which shows a blanket cylinder 15 with an alternate long and short dash line is a delocation, and shows the rotation transfer path [ in / in this drawing (c) / this blanket cylinder decondition ]. In this condition, a printing idle state or a machine is in a \*\*\*\* condition, it stuck besides the former roller 12 and the metering roller 13, and the roller 16 is also driven from the motor 20 side through the one way clutch 54, and the one way clutch 58 of another side is raced.

[0019] In drawing 4, it is the eccentric bearing which carried out eccentricity of the axis of an outer diameter, and the axis of a bore as the sign t3 in drawing showed, 60 is stuck to this eccentric bearing 60, makes a roller 16 opposite—\*\* a peripheral surface to the peripheral surface of a blanket cylinder 15, and is supported to revolve. 61 is a cam shaft, it is supported to revolve by the frame 21 on either side through the eccentric bearing 62, and it projects and the air cylinder (neither is illustrated) which while omitted illustration and projected it from the frame 21 and which rotates this cam shaft 61 through a lever is connected with the edge. Moreover, the cam 63 equipped with the periphery cam [ which projects and becomes an edge from a major diameter and a narrow diameter portion ] side projected from the frame 21 of another side of this cam shaft 61 is fixed to revolve. 65 is a roller shaft, it is fixed to the protrusion edge of the eccentric bearing 60 by the rate bundle free [ rotation accommodation ], and when becoming the eccentric section which carried out eccentricity from a ball bearing as the axial center of a shaft 65 and the sign t4 in drawing show recently, the inner ring of spiral wound gasket of 64 is being fixed.

[0020] The bolt 66 is pivoted in the frame 21 inside lobe of said eccentric bearing 62, and this bolt 66 is thrust into the stud 67 which projects from a frame 21 with the nut by the handle 68 to shaft orientations by which migration regulation was carried out. In such a configuration, by operating the air cylinder which omitted illustration and rotating a cam shaft 61, it sticks and the attachment and detachment to the blanket cylinder 15 of a roller 16 are performed. Moreover, by carrying out rotation actuation of the handle 68, it sticks and the contact pressure to the blanket cylinder 15 of a roller 16 is adjusted. 70 is the antifriction bearing of dual structure, 71 is plain bearing, and the blanket cylinder 15 is supported to revolve by the frame 21 by these antifriction bearing 70 and plain bearing 71. There are not attachment—and—detachment equipment of the sheet rotary press with which the attachment—and—detachment equipment of the blanket cylinder explained above is widely known from the former, and a place which changes exceptionally.

[0021] As shown in drawing 4, the place by which it is characterized [ of this invention ] is in the point of having set up identically the path D of a blanket cylinder 15, and the path D1 of the pitch circle of the drum gear 60, while sticking and setting up small the path d1 of the pitch circle of the clutch gear 59 to the path d of a roller 16 (d1<d), as shown in drawing 5. Thus, by constituting, it sticks during the varnish coating actuation which the rotation drive of the blanket cylinder arrival condition 15 shown in drawing 7 (b), i.e., a blanket cylinder, sticks through the clutch gear 59 and an one way clutch 58 through the middle gears 59A and 59B from the drum gear 60, and is transmitted to a roller 16, and becomes quicker than the peripheral speed of the blanket cylinder 15 which the peripheral speed of a roller 16 opposite—\*\* with this.

[0022] Therefore, when it sticks and the peripheral surface of a roller 16 always precedes rather than the peripheral surface of a blanket cylinder 15, it sticks and a skid occurs between the peripheral surface of a roller 16, and the peripheral surface of a blanket cylinder 15. For this

peripheral surface of a roller 16, and the peripheral surface of a blanket cylinder 15. For this reason, since it will stick so that it may follow and stick to rotation of a blanket cylinder 15 and may not accelerate the peripheral speed of a roller 16 and the peripheral speed of a roller 16 will not change on the boundary of measuring—area 15b of a blanket cylinder 15, and notch 15a even if it sticks with the peripheral surface of a blanket cylinder 15 in measuring—area 15b of a blanket cylinder 15 and frictional force with the peripheral surface of a roller 16 becomes large, varnish nonuniformity is reduced and quality improves.

[0023] Moreover, since it stuck and peripheral speed of a roller 16 was made quicker than the peripheral speed of a blanket cylinder 15 by setting up identically the path D of a blanket cylinder 15, and the path D1 of the pitch circle of the drum gear 60 while sticking and setting up the path d1 of the clutch gear 59 small to the path d of a roller 16, another driving source for sticking and making the peripheral speed of a roller 16 accelerate is not needed.

[0024] Drawing 8 is a model Fig. for explaining the rotation drive system in the gestalt of operation of the 2nd of the coating equipment of the printing machine concerning this invention. With the gestalt of this 2nd operation, the path of drum gear 60B is formed more greatly than the path of a blanket cylinder 15, and direct drum gear 60B is meshed to an one way clutch 58 through the middle gears 59A and 59B. 60A is another drum gear prepared on drum gear 60B and the same axle, this another drum gear 60A was formed identically to the path of a blanket cylinder 15, this another drum gear 60A has geared with the gear (not shown) of an impression cylinder 17, and rotation of a original working side is transmitted to a blanket cylinder 15 through this another drum gear 60A. Also by considering as such a configuration, the same operation effectiveness as the gestalt of the 1st operation mentioned above is acquired.

[0025] In addition, in the gestalt of this operation, although the clutch gear 59 is meshed with the blanket cylinder gear 60 by which drive connection was carried out with the original working side, it sticks and it was made to carry out the rotation drive of the roller 16, the motor which sticks and carries out independently the rotation drive of the roller 16 may be formed independently. Moreover, although it stuck [ made / it / through the one way clutch 58 ], and the gear by the side of a roller was used as the clutch gear 59, the path d1 of the pitch circle of this clutch gear 59 was stuck and it set up small to the path d of a roller 16 in order to stick the rotation by the side of a blanket cylinder 15 and to transmit to a roller 16, it is not limited to this. namely, -- if it is not necessary to mind an one way clutch 58, it sticks directly and it transmits through the roller gear 56 -- sticking -- the roller gear 56 -- sticking -- the gear by the side of a roller -carrying out -- this -- it may stick and the path of the pitch circle of a roller 56 may be set up small. Moreover, although the middle gears 59A and 59B were formed between the drum gear 60 and the clutch gear 59, the path D1 of the pitch circle of the drum gear 60 is formed more greatly than the path D of a blanket cylinder 15, and you may make it mesh the clutch gear 59 with the drum gear 60 directly. Moreover, by changing the gear ratio of the drum gear 60 and the clutch gear 59 in that case, the path D1 of the pitch circle of the drum gear 60 is made the same as that of the path D of a blanket cylinder 15, it can stick with the path d1 of the clutch gear 59, and the path d of a roller 16 can also be made the same. moreover -- although the example which twisted the blanket around the peripheral surface of a blanket cylinder 15 was explained - a plastic plate — or what twisted the plate of partial coater may be used. Moreover, although it stuck and being considered as the configuration of 3 rollers of a roller 16, the metering roller 13,

and the former roller 12, it is good also as 4 roller configurations, sticks, and can apply also to the configuration of a roller and chamber coater.
[0026]

[Effect of the Invention] As explained above, according to this invention, varnish nonuniformity is reduced and quality improves.

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#### **DESCRIPTION OF DRAWINGS**

#### [Brief Description of the Drawings]

[Drawing 1] It is the external view of the whole printing machine concerning this invention.

[Drawing 2] It is the side elevation of the coating equipment of the printing machine concerning this invention.

[Drawing 3] It is the sectional view having shown the supporting structure of the former roller in the coating equipment of the printing machine concerning this invention, and a metering roller.

[Drawing 4] It is the sectional view in which having stuck with the blanket cylinder in the coating equipment of the printing machine concerning this invention, and having shown the supporting structure of a roller.

[Drawing 5] It is the side elevation of the important section in the coating equipment of the printing machine concerning this invention.

[Drawing 6] It is a model Fig. for explaining the rotation drive system in the coating equipment of the printing machine concerning this invention.

[Drawing 7] Drawing and this drawing (c) showing a rotation transfer path [ in / in drawing for the side elevation of the important section for explaining the rotation drive in the coating equipment of the printing machine concerning this invention and this drawing (a) to explain attachment and detachment of a blanket cylinder and this drawing (b) / a blanket cylinder arrival condition ] are drawing showing the rotation transfer path in a blanket cylinder decondition.

[Drawing 8] It is a model Fig. for explaining the rotation drive system in the gestalt of operation of the 2nd of the coating equipment of the printing machine concerning this invention. [Description of Notations]

1 [ -- A metering roller, 15 / -- A blanket cylinder, 15a / -- A notch, 15b / -- Measuring area,

16 / -- It sticks and is a roller and 20. / -- A motor, 59 / -- A clutch gear, 60 / -- Drum gear. ]

-- A sheet rotary press, 5 -- Coating equipment, 12 -- A former roller, 13

#### [Translation done.]

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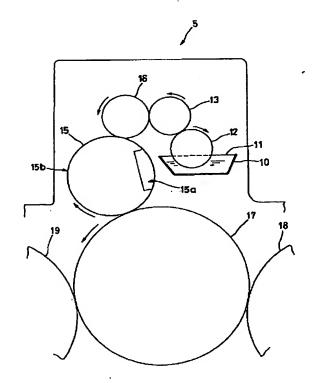
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#### (54) 【発明の名称】 印刷機のコーティング装置

#### (57)【要約】

【課題】 ニスムラを低減する。

【解決手段】 ニス舟10のニス11に浸漬された元口 ーラ12によってニス11が引き上げられ、これと対接 する調量ローラ13上に均一なニス膜が形成される。調 量ローラ13に対接する着けローラ16を介して着けロ ーラ16に対接するゴム胴15にニスが転移する。圧胴 17の爪にくわえられ搬送された紙は、ゴム胴15との 対接点を通過するときに印刷面にニスが塗布される。ゴ ム胴15は原動側と駆動連結されたゴム胴ギアを介して 回転駆動され、着けローラ16はゴム胴ギアに噛合した クラッチギアを介して回転駆動されている。そして、ゴ ム胴ギアのビッチ円の径をゴム胴の径と同一に設定する とともに、クラッチギアのピッチ円の径を着けローラの 径よりも小さく設定する。



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#### 【特許請求の範囲】

【請求項1】 コーティング装置内に設けられ、回転自 在に支持され周面に切欠きが設けられた胴と、この胴に 対接しこの胴にニスを供給する回転自在に支持された着 けローラとを備えた印刷機のコーティング装置におい て、前記着けローラの周速を前記胴の周速よりも速くし たことを特徴とする印刷機のコーティング装置。

#### 【発明の詳細な説明】

#### [0001]

【発明の属する技術分野】本発明は、印刷機の印刷ユニ ットと排紙装置との間に設けられ、印刷後の紙の印刷面 にニスを塗布して排紙装置に排出する印刷機のコーティ ング装置に関する。

#### [0002]

【従来の技術】との種の印刷機において、印刷工程を削 滅する目的で、印刷ユニットと排紙装置との間にコーテ ィング装置を設け、コーティング装置を印刷機内に組み 込んだものがある。このコーティング装置には、ブラン ケットが巻き付けられたゴム胴と、このゴム胴と対接し この対接部においてゴム胴と同一方向に同一の周速で回 20 られた胴とを備えた印刷機のコーティング装置におい 転する着けローラと、この着けローラに対接しこの対接 部において着けローラと反対方向に回転する調量ローラ とが備えられている。ゴム胴は原動側と駆動連結された ゴム胴ギアを介して回転駆動され、着けローラはゴム胴 ギアに噛合した着けローラギアを介して回転駆動されて いる。また、調量ローラは、原動側とは別のモータを駆 動源として、ニス舟のニスを引き上げる元ローラととも に回転駆動されている。

#### [0003]

【発明が解決しようとする課題】ゴム胴の周面は、ブラ 30 ンケットの両端をくわえる万力装置が収納された切欠部 と、ブランケットが巻かれたいわゆる有効面とに分けら れ、着けローラは、有効面に対接しているときと切欠部 に対接しているときとで、ニップ圧が加わった状態と抜 けた状態が繰り返されている。すなわち、着けローラは ゴム胴の有効面と対接しているときは、対接部において 同一方向に回転するゴム胴の周面との摩擦力によっても 回転駆動され、切欠部と対接しているときは、逆方向に 回転している調量ローラとの摩擦力によって制動がかけ られている。したがって、ゴム胴と着けローラ間のニッ ブ圧の管理やあるいはゴム胴の仕立て径の管理が悪くゴ ム胴の径が大きくなると、有効面においてゴム胴の周面 と着けローラの周面との摩擦力が大きくなるので、着け ローラがギアからの回転駆動ではなく、この摩擦力によ ってのみ回転駆動されることがある。

【0004】この状態においては、ゴム胴ギアに対して 着けローラギアが先行した状態となるので、ゴム胴ギア から着けローラギアへの回転駆動が遮断された状態とな る。一方、切欠部においてはゴム胴の周面と着けローラ の周面との摩擦力がほとんどなくなり、摩擦力によって 50 ている。これら元ローラ12と調量ローラ13とは、後

着けローラが回転駆動されることがなく、かつ調量ロー ラとの間の摩擦力によって減速されるので、着けローラ の回転駆動は、ゴム胴ギアと着けローラギアとの回転伝 達によって行われる。このように、胴の有効面と切欠部 との境界において、ゴム胴ギアから着けローラギアへの 回転駆動が遮断されたり連結されたりするため、互いの ギア間のバックラッシによって着けローラに大きな回転 変動が発生する。このため、紙に印刷方向と直交する方 向に帯状の縞模様のショック目が形成され、これがニス 10 ムラの原因となっていた。

【0005】本発明は上記した従来の問題に鑑みなされ たものであり、その目的とするところは、ニスムラを低 滅させた印刷機のコーティング装置を提供することにあ る。

#### [0006]

【課題を解決するための手段】との目的を達成するため に本発明は、コーティング装置内に設けられ、回転自在 に支持された着けローラと、この着けローラと対接しニ スが供給され、回転自在に支持され周面に切欠きが設け て、前記着けローラの周速を前記胴の周速よりも速くす る。したがって、常に、着けローラの周面が胴の周面よ りも先行することにより、着けローラの周面と胴の周面 との間にすべりが発生するので、仮に、胴の有効面にお いて胴の周面と着けローラの周面との摩擦力が大きくな っても、胴の回転に追従して着けローラの周速が増速さ れるようなことがなく、着けローラの周速が胴の有効面 と切欠部の境界において変化しない。

#### [0007]

ある。

【発明の実施の形態】以下、本発明の実施の形態を図に 基づいて説明する。図1は本発明に係る印刷機の全体の 外観図、図2は同じくコーティング装置の側面図、図3 は同じく元ローラと調量ローラの支持構造を示した断面 図である。図4は同じくゴム胴と着けローラの支持構造 を示した断面図、図5は同じく要部の側面図、図6は同 じく回転駆動系を説明するためのモデル図である。図7 は同じく回転駆動を説明するための要部の側面図、同図 (a) はゴム胴の着脱を説明するための図、同図(b) はゴム胴着状態における回転伝達経路を示す図、同図 (c) はゴム胴脱状態における回転伝達経路を示す図で

【0008】図1において、全体を符号1で示す枚葉輪 転印刷機には、給紙装置2と4色の印刷ユニット3と、 排紙装置4と、これら印刷ユニット3と排紙装置4との 間に設けられたコータユニットとしてのニス塗りユニッ ト5とが備えられている。図2において、ニス塗りユニ ット5には、ニス11が貯留されたニス舟10が設けら れ、このニス舟10のニス11には元ローラ12が浸漬 され、この元ローラ12には調量ローラ13が対接され

.

述するモータ20を駆動源として互いの対接部において同一方向でかつ同一の周速で回転駆動され、ニス舟10から元ローラ12によって引き上げられたニスは、調量ローラ13とのニップを通過することにより均一なニス膜として調量ローラ13の表面に形成される。

【0009】15はゴム胴であって、周面にはブランケット(図示せず)が巻き付けられているとともに、ブランケットの両端をくわえる万力装置(図示せず)を収納するための切欠き15aが設けられており、このゴム胴15の周面は、切欠き15aとブランケットが巻き付けられた有効面15bとに分けられている。ゴム胴15は原動側と駆動連結された後述するゴム胴ギア60を介して回転駆動され、このゴム胴ギア60には着けローラ16の後述するクラッチギア59が中間ギア59A、59Bを介して噛合し、着けローラ16がゴム胴15と互いの対接部において同一方向に回転している。

【0010】17はゴム胴15が対接する圧胴であって、印刷ユニット3の渡し胴18の爪にくわえられた紙が、この圧胴17の爪にくわえ替えられ搬送され、ゴム胴15との対接点を通過するときに印刷面にニスが塗布 20され、排紙装置4の排紙胴19の爪にくわえ替えられる。排紙胴19の爪にくわえられ搬送される紙は、排紙チェーンの爪竿の爪(いずれの図示を省略)にくわえ替えられ、走行する排紙チェーンによって搬送され、排紙装置4の紙積台上に落下積載される。

【0011】図3において、20はフレーム21にブラケット22を介して固定されたモータであって、モータ軸にはカップリング23を介して直結されたギアボックス24が直結され、このギアボックス24はフレーム21にブラケット25を介して固定されている。このギアボックス24にはモータ直結軸と直交して内部のベベルギアでこれと駆動連結された駆動ギア軸26が軸支されており、この駆動ギア軸26には、フレーム21から突出するスタッド27で回転自在に支持された駆動ギア28が軸着されている。

【0012】フレーム21には駆動ギア28と噛合する中間ギア29を回転自在に支持するギア軸30が軸受31を介して軸支されており、このギア軸30のフレーム21の内側突出軸受部32には元ローラ12の一端が回転自在に支持されている。元ローラ12の軸端の鍔部に40は元ローラギア33が固定され、この元ローラギア33はギア34、35を介して中間ギア29の回転が伝達されており、これらギア34、35はギア軸37の両端に軸着され、ギア軸37はフレーム21に嵌着された軸受36に回転自在に支持されている。

【0013】前記ギア軸30の軸受部32と軸受31との間には、ローラアーム39がスラストベアリングを介して遊装されており、このローラアーム39の一方の遊端部にはT字アーム40がピン41を介して揺動自在に 枢着されている。このT字アーム40の一方の遊端部に 50

は、図中符号 t 1 で示すように軸受部を偏心させた軸受 4 2 が回動調節自在に固定されており、この軸受 4 2 に は、弾性表面を有する調量ローラ 1 3 が周面を元ローラ 1 2 に対接させて軸支されている。

【0014】との調量ローラ13は、その軸端部に固定された調量ローラギア43を元ローラギア33に噛合させることにより回転駆動され、ボルトを緩めて軸受42を回動させることにより、元ローラ12に対するニップ圧が調整されるように構成されている。44は大径部と小径部を備えたカムであって、左右のフレーム21(一方のフレーム21は図示せず)間に軸架されたカム軸45の両フレーム21近接部に固定されており、このカム面には丁字アーム40の遊端部に図中符号t2で示すように偏心調節自在に枢着されたころ46が対接している。

【0015】フレーム21に植設されたスタッド47には、回動調節自在なばね軸48が一端をT字アーム40に枢着されて軸支されており、T字アーム40はばね軸48上の圧縮コイルばね49の弾発力によってころ46をカム44に圧接させる方向に回動力を付勢されている。カム軸45の図示を省略した他方のフレーム21から突出した突出端部には、レバーを介してエアシリンダのロッド(いずれも図示を省略)が枢着され、ロッドを進退させることによりカム軸45が回動し、ころ46とT字アーム40を介して元ローラ12に対し調量ローラ13が着脱されるように構成されている。

【0016】50はクラッチ軸であって、フレーム21 に固定された軸受51と、フレーム21から突出するブラケット52とによって軸支され、このクラッチ軸50上には前記駆動ギア28と 場合してモータ20側の回転をクラッチ軸50に伝達するギア53が軸着されている。クラッチ軸50上の一方向クラッチ54に固定されたカラッチギア55は、着けローラ16の軸端部に固定された着けローラギア56(図4参照)と 噛合している。この一方向クラッチ54によって、着けローラ16が被駆動側となり、モータ20側の回転のみが着けローラ16に伝達されるように構成されている。

【0017】図4に示すように、着けローラ16の軸端部にも一方向クラッチ58が装着され、これに固定された着けローラ側のギアとしてのクラッチギア59は前記ゴム胴15の軸端部に固定された胴ギア60と中間ギア59A、59Bを介して噛合している。この一方向クラッチ58によって、着けローラ16側が被駆動側となり、ゴム胴15側の回転のみが着けローラ16に伝達されるように構成されている。このように着けローラ16はモータ20側とゴム胴15側との両方から一方向クラッチ54、58を介して回転駆動されるが、両方の一方向クラッチ54、58が同時に回転を伝達することがなく、駆動側回転の速いいずれか一方の一方向クラッチが回転を伝達し遅い方が空転するように構成されている。

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のように構成することにより、図7(b)に示すゴム胴 着状態、すなわちゴム胴15の回転駆動が胴ギア60か ら中間ギア59A、59Bを介してクラッチギア59お よび一方向クラッチ58を介して着けローラ16に伝達 されるニス塗り動作中において、着けローラ16の周速 がこれと対接するゴム胴15の周速よりも速くなる。

【0022】したがって、常に、着けローラ16の周面 がゴム胴15の周面よりも先行することにより、着け口 ーラ16の周面とゴム胴15の周面との間にすべりが発 おいてゴム胴15の周面と着けローラ16の周面との摩 擦力が大きくなっても、ゴム胴 15の回転に追従して着 けローラ16の周速が増速されるようなことがなく、着 けローラ16の周速がゴム胴15の有効面15bと切欠 部15aの境界において変化しないので、ニスムラが低 滅され品質が向上する。

【0023】また、着けローラ16の径dに対してクラ ッチギア59の径d1を小さく設定するとともに、ゴム 胴15の径Dと胴ギア60のピッチ円の径D1を同一に 設定することにより、着けローラ16の周速をゴム胴1 5の周速よりも速くしたので、着けローラ16の周速を 増速させるための別の駆動源を必要としない。

【0024】図8は本発明に係る印刷機のコーティング 装置の第2の実施の形態における回転駆動系を説明する ためのモデル図である。この第2の実施の形態では、胴 ギア60Bの径をゴム胴15の径よりも大きく形成し、 中間ギア59A,59Bを介することなく、直接胴ギア 60Bを一方向クラッチ58に 噛合させたものである。 60Aは胴ギア60Bと同軸上に設けられた別の胴ギア であって、この別の胴ギア60Aはゴム胴15の径と同 一に形成され、との別の胴ギア60Aが圧胴17のギア (図示せず)と噛合しており、この別の胴ギア60Aを 介して原動側の回転がゴム胴15に伝達される。このよ うな構成とすることによっても、上述した第1の実施の 形態と同様な作用効果が得られる。

【0025】なお、本実施の形態においては、原動側と 駆動連結されたゴム胴ギア60にクラッチギア59を噛 合させ、着けローラ16を回転駆動させるようにした が、着けローラ16を独立して回転駆動するモータを別 に設けてもよい。また、ゴム胴15側の回転を着けロー ラ16に伝達するために一方向クラッチ58を介するよ ろにしたので、着けローラ側のギアをクラッチギア59 とし、このクラッチギア59のビッチ円の径d1を着け ローラ16の径dに対して小さく設定したが、これに限 定されない。すなわち、一方向クラッチ58を介する必 要がなく、直接着けローラギア56を介して伝達するの であれば、着けローラギア56を着けローラ側のギアと し、この着けローラ56のビッチ円の径を小さく設定し てもよい。また、胴ギア60とクラッチギア59との間 50 に中間ギア59A, 59Bを設けたが、胴ギア60のピ

【0018】すなわち、図7(a)において、ゴム胴1 5は実線位置が着けローラ16と圧胴17に対する着位 置であり、同図(b)はこのゴム胴着状態における回転 伝達経路を示している。 このとき、元ローラ12と調量 ローラ13とはモータ20によって駆動しており、着け ローラ16は、圧胴17、ゴム胴15側から一方クラッ チ58を介して駆動され、この状態では他方の一方向ク ラッチ54は空転している。また、同図(a)におい て、ゴム胴15は一点鎖線で示す位置が脱位置であっ て、同図(c)はこのゴム胴脱状態における回転伝達経 10 生する。このため、仮に、ゴム胴15の有効面15bに 路を示している。この状態においては、印刷停止状態あ るいは機械が緩動状態であって、元ローラ12,調量ロ ーラ13の他に着けローラ16も一方向クラッチ54を 介してモータ20側から駆動されており、他方の一方向 クラッチ58は空転している。

【0019】図4において、60は外径の軸芯と内径の 軸芯とを図中符号t3で示すように偏心させた偏心軸受 であって、との偏心軸受60には着けローラ16が周面 をゴム胴15の周面に対接させて軸支されている。61 はカム軸であって、左右のフレーム21に偏心軸受62 を介して軸支され、図示を省略した一方のフレーム21 から突出した突出端部には、レバーを介してこのカム軸 61を回動させるエアシリンダ(いずれも図示せず)が 連結されている。また、このカム軸61の他方のフレー ム21から突出した突出端部には、大径部と小径部とか らなる外周カム面を備えたカム63が軸着されている。 65はとろ軸であって、偏心軸受60の突出端部に割締 めによって回動調節自在に固定され、このころ軸65の 軸心と図中符号t4で示すだけ偏心した偏心部にはボー ルベアリングからなるころ64の内輪が固定されてい る。

【0020】前記偏心軸受62のフレーム21内側突出 部には、ボルト66が枢着されており、このボルト66 はフレーム21から突出するスタッド67にナットで軸 方向への移動規制されたハンドル68に螺入されてい る。このような構成において、図示を省略したエアシリ ンダを作動させカム軸61を回動させることにより、着 けローラ16のゴム胴15に対する着脱が行われる。ま た、ハンドル68を回動操作することにより、着けロー ラ16のゴム胴15への接触圧が調節される。70は二 重構造のころがり軸受、71は平軸受であって、これら ころがり軸受70と平軸受71とによってゴム胴15が フレーム21に軸支されている。以上説明したゴム胴の 着脱装置は、従来から広く知られている枚葉輪転印刷機 の着脱装置と格別変わるところはない。

【0021】本発明の特徴とするところは、図5に示す ように、着けローラ16の径dに対してクラッチギア5 9のビッチ円の径d 1を小さく設定(d1<d)すると ともに、図4に示すように、ゴム胴15の径Dと胴ギア 60のピッチ円の径D1を同一に設定した点にある。と

ッチ円の径D1をゴム胴15の径Dよりも大きく形成 し、胴ギア60にクラッチギア59を直接噛合させるよ うにしてもよい。また、その場合、胴ギア60とクラッ チギア59とのギア比を異ならせることにより、胴ギア 60のピッチ円の径D1をゴム胴15の径Dと同一に し、クラッチギア59の径d1と着けローラ16の径d とを同一にすることもできる。また、ゴム胴15の周面 にブランケットを巻き付けた例を説明したが、樹脂凸版 やあるいは部分コータの版材を巻き付けたものでもよ い。また、着けローラ16、調量ローラ13、元ローラ 10 ける回転駆動を説明するための要部の側面図、同図 12の3本ローラの構成としたが、4本ローラ構成とし てもよく、着けローラとチャンバーコータの構成にも適 用できる。

#### [0026]

【発明の効果】以上説明したように本発明によれば、ニ スムラが低減され品質が向上する。

#### 【図面の簡単な説明】

【図1】 本発明に係る印刷機の全体の外観図である。

【図2】 本発明に係る印刷機のコーティング装置の側 面図である。

【図3】 本発明に係る印刷機のコーティング装置にお ける元ローラと調量ローラの支持構造を示した断面図で\* \*ある。

【図4】 本発明に係る印刷機のコーティング装置にお けるゴム胴と着けローラの支持構造を示した断面図であ

【図5】 本発明に係る印刷機のコーティング装置にお ける要部の側面図である。

本発明に係る印刷機のコーティング装置にお 【図6】 ける回転駆動系を説明するためのモデル図である。

【図7】 本発明に係る印刷機のコーティング装置にお (a) はゴム胴の着脱を説明するための図、同図(b)

はゴム胴着状態における回転伝達経路を示す図、同図 (c) はゴム胴脱状態における回転伝達経路を示す図で

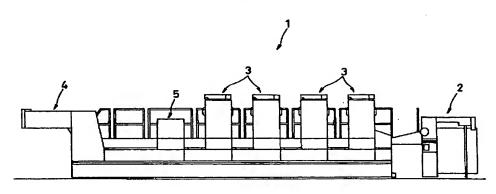
【図8】 本発明に係る印刷機のコーティング装置の第 2の実施の形態における回転駆動系を説明するためのモ デル図である。

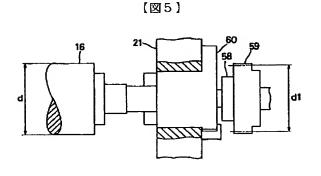
#### 【符号の説明】

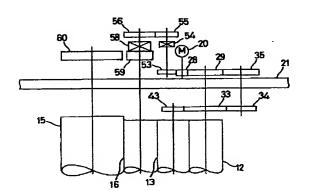
ある。

1…枚葉輪転印刷機、5…コーティング装置、12…元 20 ローラ、13…調量ローラ、15…ゴム胴、15a…切 欠き、15 b…有効面、16…着けローラ、20…モー タ、59…クラッチギア、60…胴ギア。

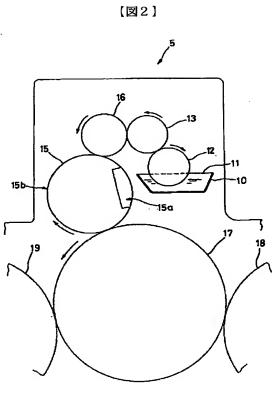
【図1】

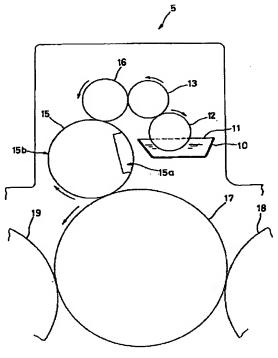


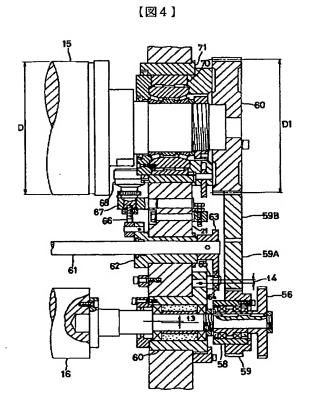


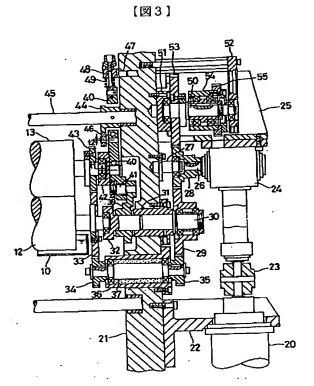


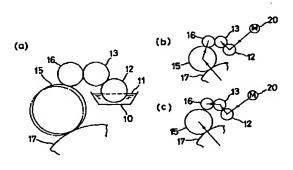
【図6】



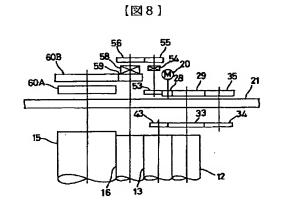








[図7]



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